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- = - = , where 2 ) 2 ( 2 L C

= (3.4.4) Substituting (3.4.4) into

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(3.4.2) we have,  $22aCaAellp - =$

We can find the optimum solution by  $0 = \frac{a}{A}$ , by some operations leads to  $0^2 - 22^2 = -$   
 $- = \frac{aC}{aC} \frac{a}{A}$ , therefore we have  $2^2 C a =$  for  $0, > ba$

Substitute it back to (3.4.4), we have  $a C b = = 2^2$  (3.4.5) That

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means the optimum cross-section  
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end by a linear spring with spring constant  $k$ . Find the equation for buckling loads. Hint: The boundary conditions are  $0 = w$  and  $0 = \frac{dw}{dx}$  at the left end; and  $0 = M$  and  $0 = V - k w$  at the right end.

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the opposite side as we assumed.--- ANS (b) Figure 5.32 (1) Since this cross-section is symmetric with respect to y axis, shear center is located in the y axis.

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